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Shinjo

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[54] RECESSED SCREW

5,435,680 7/1995 Schuster 411/404

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164318 12/1985 European Pat. Off. 411/403

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Attorney, Agent, or Firm—Antonelli, Terry, Stout & Kraus

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[57] ABSTRACT

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[52] U.S. Cl. **411/404; 411/919; 81/436; 81/460**

[58] Field of Search **411/402, 403, 411/404, 405, 919; 81/436, 460**

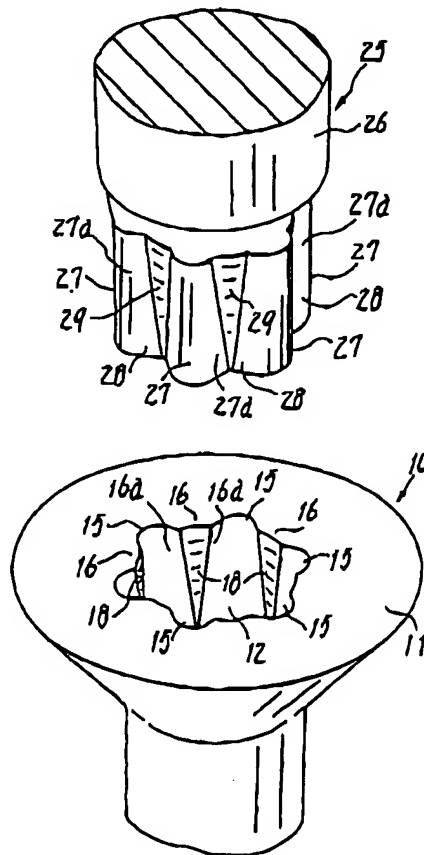
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A recessed screw has a recess formed in the head of screw, with the recess being defined as a superimposition of two equilaterally-triangular recesses each having vertices shaped arcuate and arranged coaxially with a phase shift of 60° around the screw axis. The recess thus has alternating six grooves and six driven vanes having side walls extending parallel with the axis, and an inner ridge of vane is a conically curved sucking wall inwardly slanted towards a bottom of the recess. A cross-sectional area of groove is larger than that of each vane. A driver bit has six driving vanes in conformity with the grooves and six valleys each interposed between the adjacent driving vanes and corresponding to the driven vanes in the recess, such that a biting action appears between the recess and bit, the driving vanes can be strengthened, and any conventional bit for Torx screws can be used with the recessed screw.

4 Claims, 4 Drawing Sheets



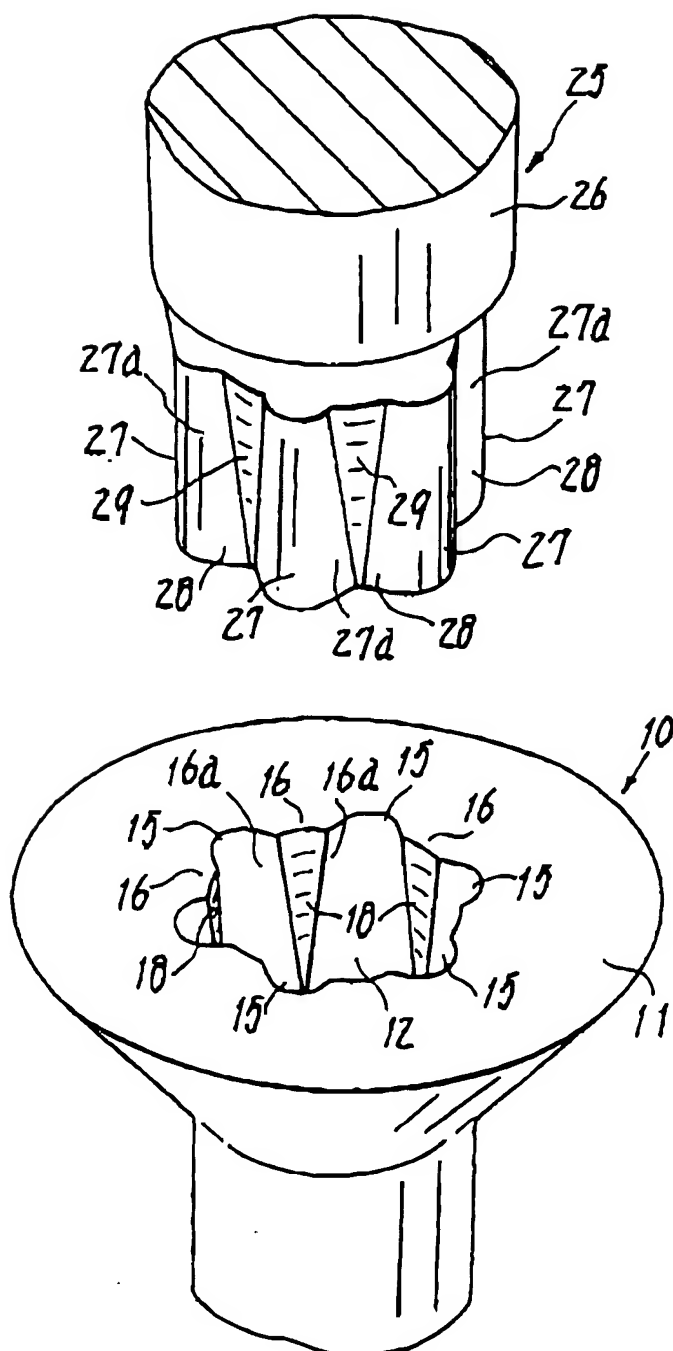
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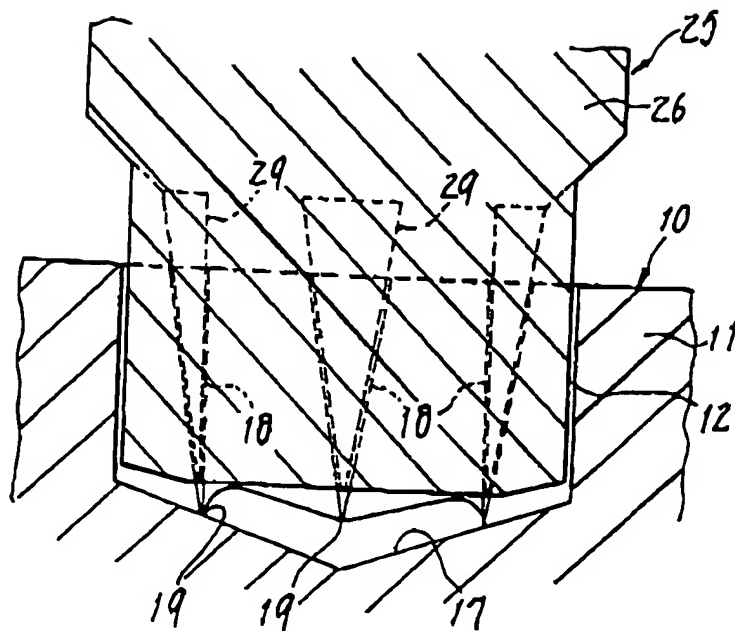
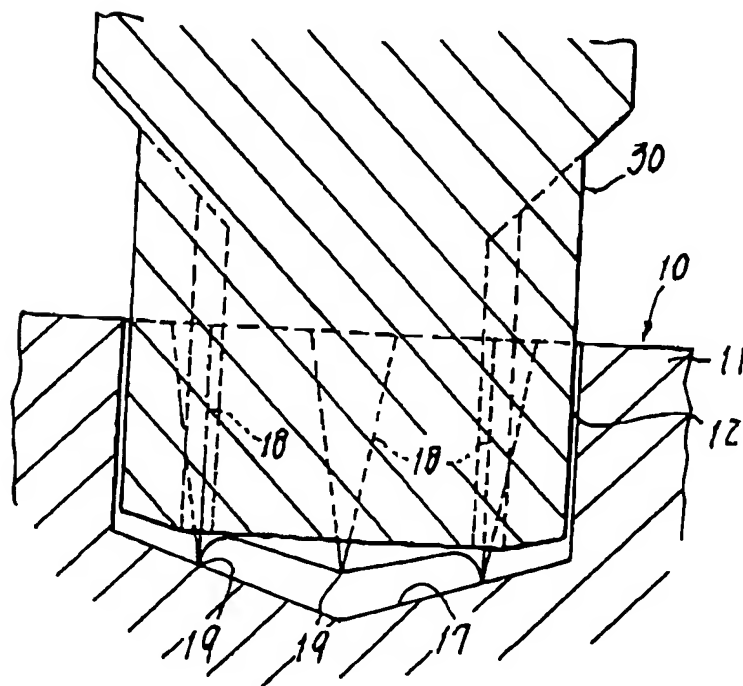
Sep. 10, 1996

Sheet 2 of 4

5,553,983

FIG. 3



U.S. Patent**Sep. 10, 1996****Sheet 3 of 4****5,553,983****FIG. 4****FIG. 5**

U.S. Patent

Sep. 10, 1996

Sheet 4 of 4

5,553,983

FIG. 6
PRIOR ART

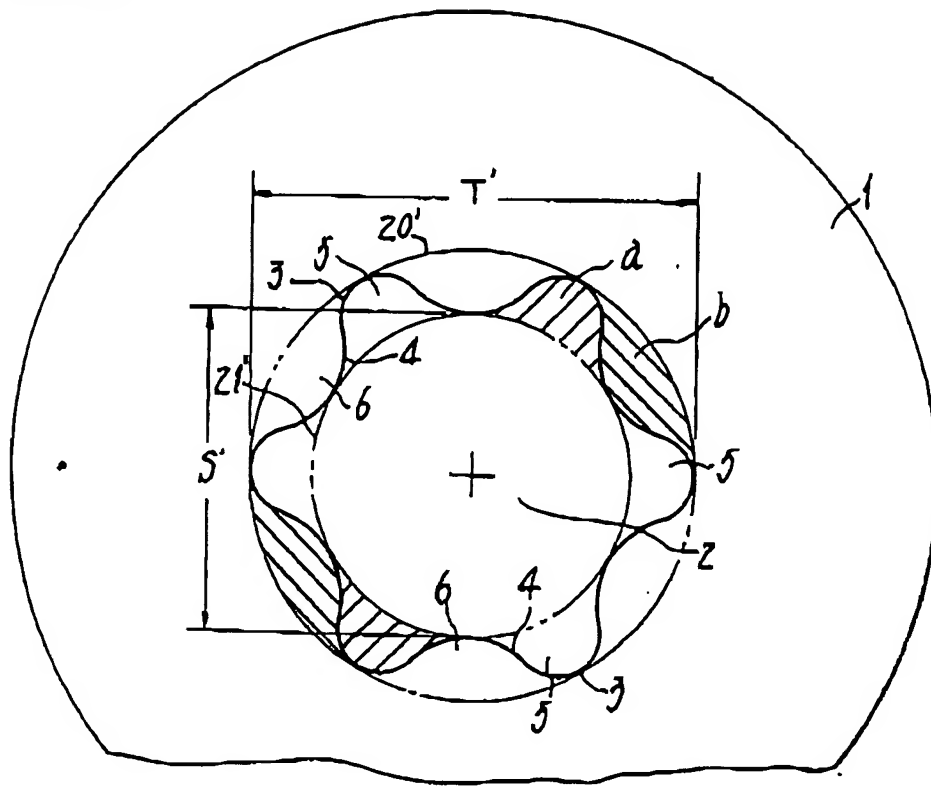
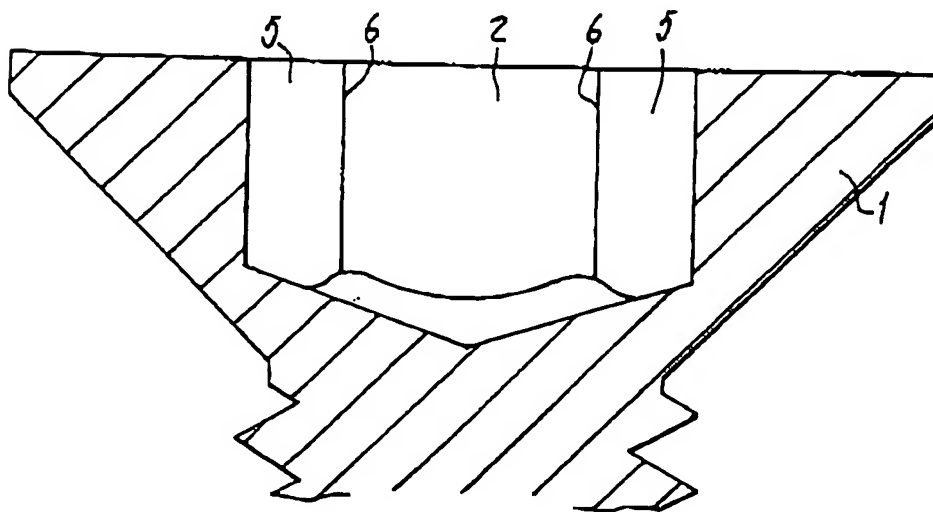


FIG. 7
PRIOR ART



5,553,983

1

RECESSED SCREW

BACKGROUND OF THE INVENTION

1. Field of the Invention

The present invention pertains a recessed screw having a recess formed in the head thereof to receive a driver bit, more particularly relates to an improved shape of the recess, and further includes the driver bit for use with and engageable with the recessed screw having the improved recess.

2. Description of Prior Art

One of the prior art, recessed screws which is called a "Torx" (a registered trademark, see Japanese Utility Model Publication No. Sho. 48-39959) does comprise a screw head 1 in which a recess 2 engageable with a driver bit is formed as shown in FIGS. 6 and 7. This recess is defined by semicylindrical concaves 3 and semicylindrical convexes 4 which alternate one with another, and each convex continues smoothly to the adjacent concaves. The concaves 3 are retracted radially and outwardly, with the convexes 4 protruding radially and inwardly so that they provide six grooves 5 and six thick vanes 6 to be driven. The side driven walls of each vane extend straight in parallel with an axis of the screw. Because of this feature, the Torx screws are advantageous in that there arises no problem of "ride-out" of the driver bit when it is driven, thus permitting a high torque to be applied to the recessed screw.

Contrary to usual screws each having a crossed recess, the Torx screws which lack the "biting" action for driver bits are disadvantageous in the automatic fastening of the screws. A clearance between the recess of Torx screw and the driver bit is so large that the bit is likely to slip off the recess. This is a serious problem in particular when a self-drilling screw or an elongate self-tapping one is fastened at high rotational speeds by a vibrating bit.

In general, the self-drilling and self-tapping screws are hardened before delivery to users. Thus, the driven vanes in their recesses are rendered harder and of a significantly higher mechanical strength as compared with non-hardened screws. On the other hand, the driver bit has driving vanes designed for engagement with the grooves 5 in the recess 2 of Torx screws. A cross-sectional area of the driving vane is therefore designed to match such an area "a" of groove 5, which however is much smaller than that "b" of each driven vane 6. Consequently, the driver bit will be worn soon if repeatedly used with the hardened screws of the self-drilling or self-tapping types.

As noted above, the side wall of each driven vane 6 in the Torx screw recess 2 is composed of the semicylindrical concave 3 and convex 4 continuing one to another. Therefore, it is difficult for conventional instruments to accurately measure the dimensional precision of such curved walls. Since only a minute clearance is allowed between the recess 2 of a smaller screw and a driver bit fitting therein, it is not easy to ensure a satisfactory preciseness to those recess 2 and bit.

SUMMARY OF THE INVENTION

The present invention was made in view of the problems inherent in the prior art Torx screws, and an object of the present invention is therefore to provide a recessed screw having a recess which surely bites a driver bit fitted therein, offers a possibility of remarkably raising the strength of driving vanes of the bit, and also another possibility of being used in combination with the prior art driver bit for Torx

2

screws. The invention further provides a driver bit of a novel type designed for use with the recessed screw proposed herein.

In order to achieve these objects, a recessed screw in accordance with the present invention comprises a head and a recess formed therein, characterized in that the recess is defined as a coaxial superimposition of two equilateral-triangular and cylindrical bores which each have arcuate vertices and are arranged with a phase shift of 60° around an axis of the screw to thereby provide six engageable grooves and six driven vanes alternating one with another, wherein side walls of each driven vane extend in parallel with the axis, and inner ridges of the driven vane are chamfered each to form a conically curved sticking wall which is inwardly slanted towards a bottom of the recess. In this recessed screw, the cross-sectional area of each engageable groove is larger than that of each driven vane.

A driver bit which is provided herein and adapted for use to fasten the recessed screw as summarized above is characterized by six driving vanes for engagement with the engageable grooves of the recess formed in the screw, such that six valleys are formed each between the adjacent driving vanes so as to fit on the respective driven vanes of said recess.

When the screw is fastened by the driver bit fitted in the recess of the screw, bottoms of the valleys formed on said bit will tightly engage with the conically curved sticking walls in the recess whereby a biting action is produced for the driver bit. With a fastening torque being imparted to said bit to this state, the side surface of each driving vane will come into close contact with the side surface of each driven vane of the recessed screw so that the torque is transmitted from the former to the latter. No "ride-out" of bit will occur during this operation, because all the side surfaces of driving and driven vanes extend in parallel with the axis of screw. Since each groove in the recess is substantially larger in cross-sectional area than each driven vane, the cross-sectional area of each driving vane of the bit can be made equal to that of said groove in the recess. This feature is advantageous in that the mechanical strength of driver bit is improved to a remarkable degree as compared with the prior art driver bit for Torx screws.

BRIEF DESCRIPTION OF THE DRAWINGS

FIG. 1 is a plan view of a recessed screw provided in the present invention;

FIG. 2 is a vertical cross section in part of the recessed screw;

FIG. 3 is a perspective view showing in part a driver bit together with the recessed screw;

FIG. 4 is a vertical cross section of the driver bit in engagement with a recess in the screw;

FIG. 5 also is a vertical cross section of a prior art driver bit in engagement with the recess;

FIG. 6 is a plan view of a prior art recessed screw; and

FIG. 7 is a vertical cross section in part of the prior art screw.

DETAILED DESCRIPTION OF THE PREFERRED EMBODIMENTS

The embodiments of the present invention will now be described below referring to the drawings, in which FIGS. 1 and 2 show a recessed screw provided herein and FIG. 3 illustrates the screw in engagement with a driver bit.

5,553,983

3

A recessed screw in this embodiment comprises a head 11 having a recess 12 formed therein for engagement therewith. The recess has in cross section a contour defined by superimposing two equilaterally-triangular bores 14 and 14 coaxial with one another. Each bore has vertices each shaped arcuate to give a partially cylindrical surface 13, and those bores are arranged symmetrically with a phase shift of 60° around an axis "O—O" of the screw so that six engagable grooves 15 and six driven vanes 16 are formed. These grooves and vanes alternate one with another, and side walls 16a of each driven vane 16 are flat planes each continuing to the partially cylindrical surface 13 of the arcuate vertex 13. The cylindrical surface is a bottom of each engagable groove 15, and the side walls 16a extend in parallel with the axis "O—O" and reach a bottom 17 of this recess 12. The side walls 16a intersect one another to provide an inner ridge of the driven vane 16. Each ridge is chamfered to provide a sticking wall 18 which is conically curved in cross section and inwardly slanted at a gentle angle (of or less than about 10°) towards the bottom 17 of the recess 12. In detail, the sticking walls 18 which are intended to bite an inserted driver bit 25 as described below are sections of a conical surface having its center aligned with the axis "O—O" and being inwardly slanted towards the bottom 17 of said recess. Transverse width of each sticking wall 18 gradually decreases from its upper end towards its lower end 19, which merges into an intersecting point of two side walls 16a, thus converging into the bottom 17.

Mechanical strength of each driven vane 16 depends on, and substantially in proportion to, a cross-sectional area "B" surrounded by a circumcircle 20 of recess 12, the side walls 16a and partially cylindrical surface 13. In the recessed screw provided herein, the cross-sectional area "B" is smaller than that "A" of each engagable groove 15. The latter cross-sectional area "A" is for a space which is defined and surrounded by an inscribed circle 21, the side walls 16a and partially cylindrical surface 13. The inscribed circle 21 includes every nodal line on which extrapolations of the adjacent side walls 16a intersect one another. This means that the driver bit 25 can have, for engagement with the grooves 15, driving vanes 27 each of a cross-sectional area larger than that "B" of each driven vane 16 in the recess 12. This feature makes it possible to strengthen the driver bit 25 to be more durable.

FIGS. 3 and 4 show the driver bit 25 which is engagable with the recess 12 so as to fasten the described screw 10. The driver bit comprises six driving vanes 27 ridge-shaped and formed at an end of a shank 26, in conformity with the grooves 15 in the recess 12. The driver bit further comprises six valleys 28 each disposed between the adjacent driving vanes 27, also in conformity with the driven vanes 16 in said recess 12. A bottom of each valley 28 is a conically curved surface 29 which can closely fit on the conically curved sticking wall 18 present in the recess 12.

In use, the bit 25 will be inserted at first in the recess 12 of the screw 10 so that conically curved bottom surfaces 29 of the former come into close engagement with the conically curved sticking walls 18 of the latter, thus the biting action appearing between them. With a fastening torque being subsequently applied to the bit 25, one of side walls 27a of each driving vane 27 will contact the corresponding side wall 16a of each driven vane 16 in the recess 12 so as to transmit the torque through the latter wall 16a to the screw 10. Since the torque-transmitting walls 16a and 27a are formed parallel with the axis "O—O" of screw 10 and bit 25, respectively, the torque transmitted from the bit does not produce any axial component but imparts only a rotational

4

moment to the screw. Thus, the driver bit 25 will not be caused to "ride out" of the recess while fastening the screw, but efficiently transmit the torque thereto. Even if the driver bit 25 is rotated at considerably high speeds, it will never slip off the recess 12.

A ratio S/T of the diameter "S" of inscribed circle 21 to the diameter "T" of circumcircle 20 for the recess 12 is one of important parameters. The diameter ratio S/T may be designed to be the same as that S'/T' for the diameter S' of an inscribed circle 21' and the diameter T' of a circumcircle 20' of the recess 2 in the prior art Torx screw shown in FIG. 6. In such a case, a driver bit 30 for Torx screws can also fit in the recess 12 to drive the screw 10, as shown in FIG. 5. Usually, the diameter ratio S'/T' fall within a range from 0.70 to 0.75 for the Torx screws.

As will be best seen in FIG. 1, the side walls 16a and 16a of the adjacent driven vanes 16 in the recess 12 lie in parallel with each other. A distance "E" between the walls 16a can be measured easily with any conventional instrument, for a better control of the precision of recesses 12. The same applied also to the driver bits.

In summary, the recessed screw offered herein is satisfactory in its biting action for the inserted driver bit, which can now be driven at higher speeds, without any fear of slipping off the screw and without the problem of "ride-out". Therefore, the screw can be fastened more efficiently with a higher torque. As for the driver bit, the cross-sectional area of its driving vanes can be made so larger than that of the driven vanes of the screw that the strength and durability of the bit is remarkably improved.

The structures of recessed screw and driver bit which the present invention provides are advantageous in particular for the self-drilling or self-tapping screws rotated at extremely high speeds to transmit a stronger torque.

What is claimed is:

1. A recessed screw comprising a screw body having a head with a recess formed therein, the recess being defined as a coaxial superimposition of two equilaterally-triangular recesses which each have arcuate vertices and which are arranged with a phase shift of 60° about a longitudinal axis of the recessed screw to thereby provide six engagable grooves and six driven vanes alternating with one another at regular angular intervals, inner ridges of each of said driven vanes being chamfered to form a conically curved sticking wall which is inwardly slanted towards a bottom of the recess, with sidewalls of each driven vane converging to form a point at the inner end of the sticking wall as the sticking wall intersects the bottom of the recess, the sidewalls of each driven vane extending in parallel with the longitudinal axis of said screw to provide flat planes continuing to a partially cylindrical surface of the corresponding arcuate vertex, the cross-sectional area of each engagable groove being larger than the cross-sectional area of each of the adjacent driven vanes over at least the inner half of the recess, each cross-sectional area being defined by a first circle inscribing innermost points of said six driven vanes and a second circle circumscribing the arcuate vertices of said six engagable grooves.

2. A recessed screw in accordance with claim 1, wherein the ratio of the diameter of the inscribed circle to the diameter of the circumscribed circle is in the range of from 0.70 to 0.75.

3. A recessed screw comprising a screw body having a head with a recess formed therein, said recess being defined as a coaxial superimposition of two equilaterally-triangular recesses which each have arcuate vertices and which are arranged with a phase shift of 60° about a longitudinal axis

5,553,983

5

of the recessed screw to thereby provide six engagable grooves and six driven vanes arranged at regular angular intervals, each engagable groove having an arcuate outer bottom and inner sidewalls, with adjacent sidewalls of each pair of adjacent engagement grooves defining one of said six driven vanes, the sidewalls of each driven vane extending in parallel with the longitudinal axis of the recessed screw, and inner ridges of each driven vane being chamfered to form a conically curved sticking wall which is inwardly slanted towards a bottom of the recess, wherein over at least the

6

inner half of the recess the cross-sectional area of each engagable groove is larger than the cross-sectional area of the adjacent driven vane.

4. A recessed screw in accordance with claim 3, wherein the ratio of the diameter of a circle inscribing the innermost points of said six driven vanes to the diameter of a circle circumscribing the arcuate vertices of said six engagable grooves is in the range of from 0.70 to 0.75.

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